



DTX-LG

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DTX-LG



LINEAR SYSTEMS

1.1 Features of DTX MOTION Linear Guide

● High Accuracy

Linear Guide has little friction, only a small driving force is needed to move the load. Low friction helps the temperature rising effect to stay low, Thus, the friction is decreased and the accuracy can be maintained for a long period than traditional slide system.

● High Rigidity

The design of Linear Guide features an equal load rating in all directions that provide sufficient rigidity load in all directions, self-aligning capability to absorb installation error. Moreover, a sufficient preload can be achieved to increase rigidity and makes it suitable for any kind of installation.

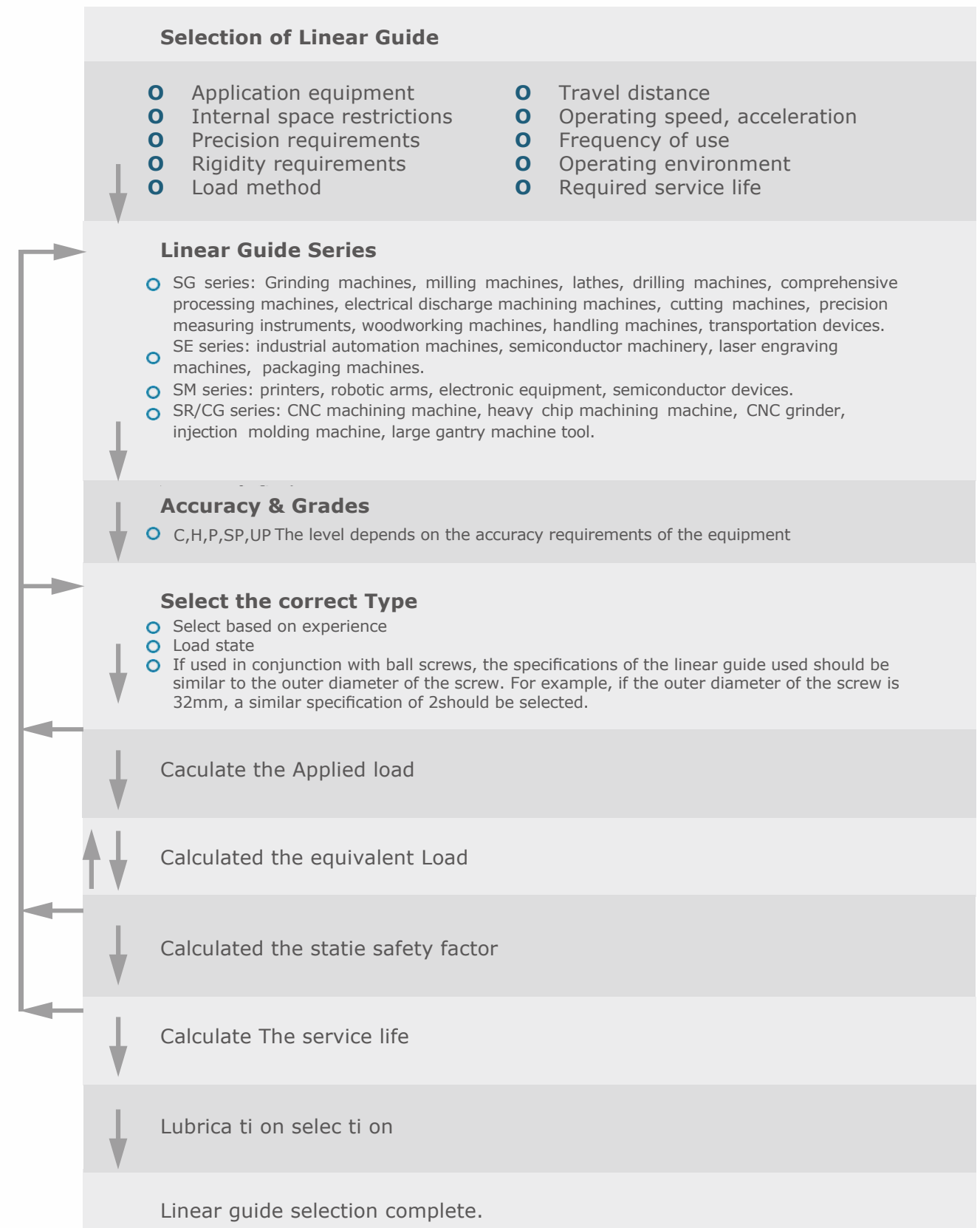
● Easy for Maintenance

Compared with high-skill required scraping process of traditional slide system, the Linear Guide can offer high precision even if the mounting surface is machined by milling or grinding. Moreover, the interchangeability of Linear Guide gives a convenience for installation and future maintenance.

● High Speed

Linear Guide block, rail and ball apply by contact point of rolling system. Due to the characteristic of low friction, the required driving force is much lower than that in other systems, thus the power consumption is low, Moreover the temperature rising effect is lower even under high speed operation.

1.2 The Procedure of Select Linear Guide



Basic Load rating and Service Life of Linear Guide

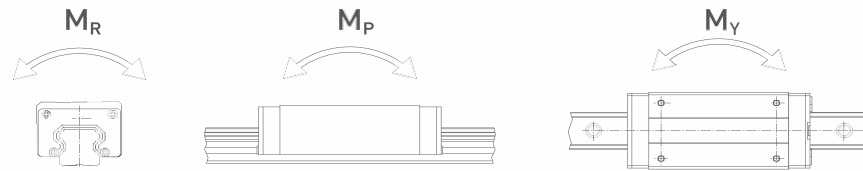
When determining a model that would suit your service conditions for a linear motion system, the load carrying capacity and service life of the model must be considered. To consider the load carrying capacity you should know the static safety factor of the model calculation based on the basic static load rating. Service life can be assessed by calculating the nominal life based on the basic dynamic load rating and checking to see if the obtained value meet your requirements.

The service life of a linear motion system refers to the total running distance that the linear motion system travels until flaking (the disintegration of a metal surface in scale-like pieces) occurs there to as a result of the rolling fatigue of the material caused by repeated stress on raceways and rolling elements.

Basic Load Rating : There are two basic load ratings for linear motion systems : basic static load rating (Co), which sets the static permissible limits, and basic dynamic load rating (C).

1.3 Basic Permissible Moment (MX, MY, MZ)

When a Linear Guide gets a force that makes the balls distorted to 1/10,000 of their diameter, we call the force as basic static permissible moment. Values of MX, MY, MZ are shown on Fig 1.3.1, which suggest 3 axis of moment on a Linear Guide slide



Static safety factor

When linear guides are used in slow motion or low-frequency applications, the static safety factor must be considered. Depending on the specific application conditions, different safety factors must be applied when calculating the static load, especially when the guide is subjected to impact loads, in which case a larger safety factor should be used.

Table | Static Safety Factor Usage

Load conditions	f_{sl} , f_{sm} lower limit
General operating conditions	1.0~3.0
Impact and vibration during operation	3.0~5.0

$$f_{SL} = \frac{C_0}{P} \quad \text{or} \quad f_{SM} = \frac{M_0}{M} \quad \dots \dots \dots \text{Eq.1.1}$$

$$f_s = \frac{C_0}{P} \cdot \frac{M_0}{M}$$

f_s : static safety factor
 C_0 : basic static load rating (kg)
 M_0 : static permissible moment (kg-mm)
 P : calculated load (kg)
 M : calculated moment (kg-mm)

A linear motion system may possibly receive an unpredictable external force due to vibration and impact while it is at rest or is moving or due to inertia resulting from start and stop. It is therefore necessary to consider the static safety factor against operating loads like these. The static safety factor (f_s) indicates the ratio of a linear motion system load carrying capacity (basic static load rating C_0) to the load exerted there on.

To calculate the applied load on Linear Guide, mean load and static safety factor must be obtained in advance. In the working environment with high intensity while start and stop, cantilever or cutting, an unexpected heavy load may occur. therefore the maximum load must be acquired. Datum values of static safety factor are shown below,

1.4 Service Life

1.4.1 Service Life (L)

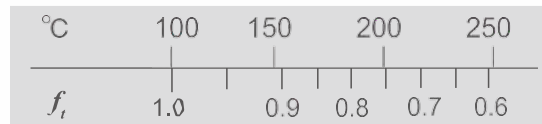
Even when identical linear guideways in a group are manufactured in the same way or applied under the same condition, the service life may be varied. Thus, the service life is used as an indicator for determining the service life of a linear guideway system. The nominal life (L) is defined as the total running distance that 90% of identical linear guideways in a group, when they are applied under the same conditions, can work without developing flaking.

1.4.2 Basic Dynamic Load Rating (C)

(2) 【f_t:Temperature factor】

For Linear Guide used at ambient temperatures over 100°C, a temperature factor corresponding to the ambient temperature, selected from the diagram below, must be taken into consideration. In addition, please note that selected Linear Guide itself must be a model with high-temperature specifications.

Temperature



(3) 【f_w:Load factor】

In general, machines in reciprocal motion are likely to cause vibration and impact during operation, and it is particularly difficult to determine the magnitude of vibration that develops during high-speed operation as well as that of impact during repeated starting and stopping in normal use. Therefore, where the effects of speed and vibration are estimated to be significant divide the basic dynamic-load rating (C) by a load factor selected from the table below.

Table2 Load Coefficient

Load condition	Operating speed	f _w
No impact force and smooth	V≤15m/m in	1-1.2
Minimal impact force	15m/m in<V≤60m/m in	1.2-1.5
Normal load force	60m/m in<V≤120m/m in	1.5-2.0
Subject to impact force and vibration	V>120m/m in	2.0-3.5

1.4.5 Conversion of service life into time

Convert service life distance into service life time based on operating speed and frequency

$$L_h = \frac{L \cdot 10^3}{V_e \cdot 60} = \left(\frac{C}{P} \right)^3 \cdot \frac{50 \cdot 10^3}{V_e \cdot 60} \text{ hr} \quad \dots \dots \dots \text{Eq.1.4}$$

L_h: Service life time (hr)

L: Service life (km)

V_e: Operating speed (m/min)

C/P: Load ratio

1.5 Workload

1.5.1 Workload Calculation

The load applied to the Linear Guide, varies with the external force exerted thereon, such as the location of the center of gravity of an object been moved, the location of the thrust developed, inertia due to acceleration and deceleration during starting and stopping, and the machining resistance. To select the correct type of Linear Guide, the magnitude of applied loads must be determined in consideration of the above conditions to calculate accurate applied load.

To obtain the magnitude of an applied load and the service life in hours, the operating conditions of the Linear Guide system must first be set.

(1) Load on a single slider

Table 3 Load Calculation Example

Linear guide configuration diagram	Force distribution diagram	Load on a single slider

(2) Inertial force load

Table 4 Calculation Example of Inertial Force Load

Example of considering acceleration and deceleration	The force borne by a single slider
F: Drive thrust (N) W: Device weight (N) g: Gravitational acceleration (9.8 m/sec²) Pn: Load (radial, reverse radial) n=1~4 V: Speed t₁: Time constant n = 1 ~ 3 c, d, l: Distance	- Isokinetic $P_1 \sim P_4 = \frac{W}{4}$ - accelerate $P_1 = P_3 = \frac{W}{4} + \frac{1}{2} \cdot \frac{W}{g} \cdot \frac{V_c}{t_1} \cdot \frac{l}{d}$ $P_2 = P_4 = \frac{W}{4} - \frac{1}{2} \cdot \frac{W}{g} \cdot \frac{V_c}{t_1} \cdot \frac{l}{d}$ - decelerate $P_1 = P_3 = \frac{W}{4} - \frac{1}{2} \cdot \frac{W}{g} \cdot \frac{V_c}{t_3} \cdot \frac{l}{d}$ $P_2 = P_4 = \frac{W}{4} + \frac{1}{2} \cdot \frac{W}{g} \cdot \frac{V_c}{t_3} \cdot \frac{l}{d}$

1.5.2 Average load calculation

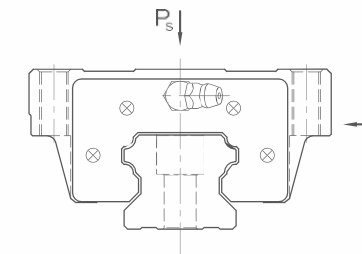
The load borne by the slider during operation is sometimes not equal. For example, in the operation of a conveying device, when it moves forward, it bears the additional weight of the goods, and when it returns, it only bears the weight of the device itself. The load changes in a stepped manner, so the average load during operation must be calculated to calculate the service life. The definition of average load is the equivalent load value that is equal to the lifespan under load variation conditions.

Table 5 Calculation Example of Average Load

Types of load changes	Types of load changes
Step by step change 	$P_m = \sqrt[3]{1/L(P_1^3 \cdot L_1 + P_2^3 \cdot L_2 + \dots + P_n^3 \cdot L_n)}$ P_m: Average load P_n: Variable load L: Total running distance L_n: Operating distance under P_n load
Monotonic variation 	$P_m = 1/3 (P_{min} + 2 \cdot P_{max})$ P_m: Average load P_{min}: Minimum load P_{max}: Maximum load
Sinusoidal variation 	$P_m = 0.65 \cdot P_{max}$ P_m: Average load P_{max}: Maximum load

1.5.3 Calculation of Equivalent Load in Two Directions

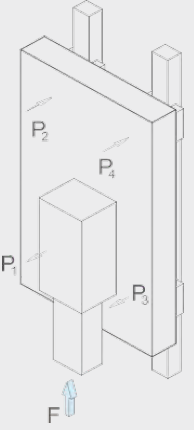
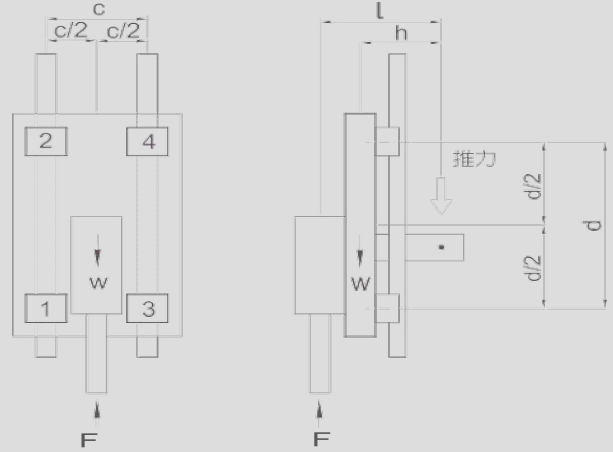
DTX-LG linear guides can withstand loads in four directions: up, down, left, and right. Therefore, when using linear guides, it is possible to be subjected to both vertical loads and lateral loads imultaneously. The equivalent load can be calculated using the following formula.



SG/SE/SM/SR/ series

$$P_e = P_s + P_l \quad \dots \dots \dots \text{Eq.1.5}$$

Table 6. Calculation Example of Service Life

Specifications for the use of linear guides	Device dimensions	Condition
Type: SGH 30CA C:38.74kN C ₀ :52.19kN Preload: Z0	d:600mm c:400mm h:200mm I:250mm	Equipment weight (W): 15 kN Drilling force (F): 1 kN System temperature: Room temperature Load condition: Normal load
 		
- Calculation of load bearing capacity of slider $P_1 \sim P_4 = + \frac{W \cdot h}{2d} - \frac{F \cdot l}{2d} = + \frac{15 \times 200}{2 \times 600} - \frac{1 \times 250}{2 \times 600} = 2.29 \text{ (kN)}$ $P_{\max} = P_1 \sim P_4 = 2.29 \text{ (kN)}$ - Since Z0 preload is selected, $P_c = P_{\max} = 2.29 \text{ (kN)}$ Note: Choosing heavier preloads (such as Billion A and ZB) may increase rigidity, but it will reduce its service life. Calculation of lifespan L. $L = \left(\frac{f_h \times f_t \times C}{f_w \times P_c} \right)^3 \times 50 = \left(\frac{1 \times 1 \times 38.74}{2 \times 2.29} \right)^3 \times 50 = 30,258 \text{ (km)}$		

1.6 Friction

The construction of Linear Guide are block, rail and motion system which has rolling elements, such as balls and rollers, placed between two raceways. The rolling motion that rolling elements give rise to reduce the frictional resistance to 1/20 th to 1/40 th of that in a slide guide. Static friction, in particular, is much lower in a linear motion system than in other system, and there is little difference between static and dynamic friction, so that stick-slip does not occur. Therefore, Linear Guide could apply in various precision motion system. Frictional resistance in a linear motion system varies with the type of linear motion system, the magnitude of the preload, the viscosity resistance of the lubricant used the load exerted on the system, and other factors. Table shows Friction of Linear Guide.

$$F = \mu \cdot W + S \quad \text{Eq.1.8}$$

F: Friction force (kN)
 S: Resistance of oil scraper (kN)
 μ : Coefficient of friction
 W: Vertical load in motion (kN)

1.7 Lubrication

Lubrication is essential to linear motion system. Without lubrication, the friction of rolling parts increases and might be the main factor of service life shortening.

A Lubricant :

- (1) Reduces friction on moving parts, thereby to prevent wearing due to raise in temperature.
 - (2) Forms an oil film on rolling surfaces, thus decreasing stress that develops on the surfaces and safeguarding the system against rolling fatigue.
 - (3) Covers metal surfaces with an oil film, thereby preventing rust.
- To tap the full function of a linear motion system, lubrication is essential to meet the system service conditions.

Classification of Lubricants

Primarily grease and sliding surface oil are used as lubricants for linear motion systems. In general a lubricant must :

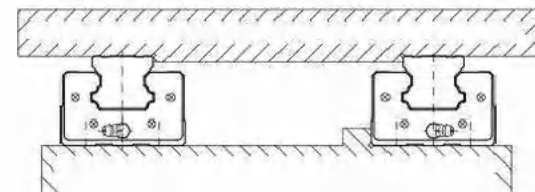
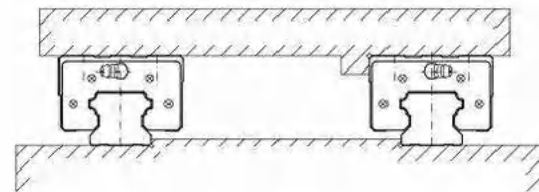
1.9 Linear Guide Configuration

Linear guides can withstand loads in the up, down, left, and right directions, so linear guide assemblies can be configured according to the machine structure and working load direction.



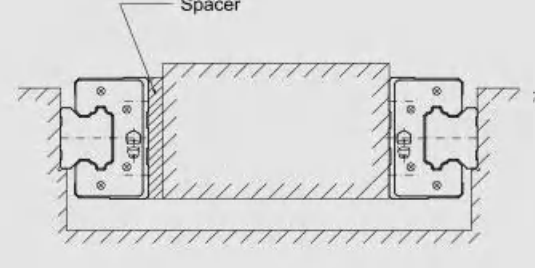
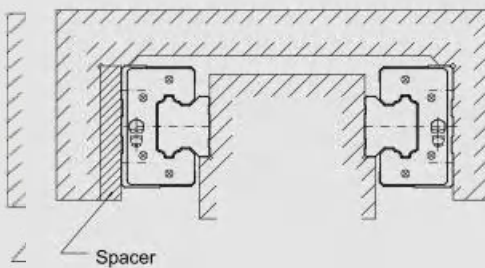
Horizontal Configuration.

Upside down Configuration.

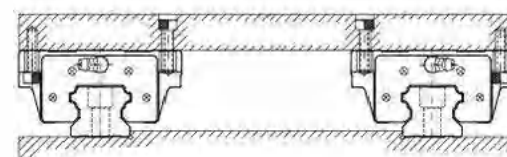
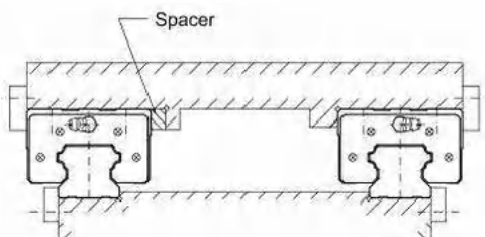


Vertical Configuration.

Back-to-back configuration



Common Fastening Method of Linear Guide

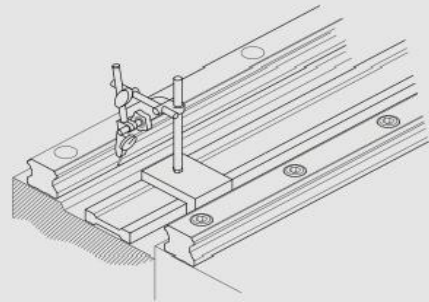


A Setting Where the Host Machine is Subjected to Impact and Vibration Common

1.10 Linear Guides Installation

Linear guides must be installed according to the operating conditions of the machine, such as the degree of vibration and impact force, the required Traveling accuracy, and machine restrictions.

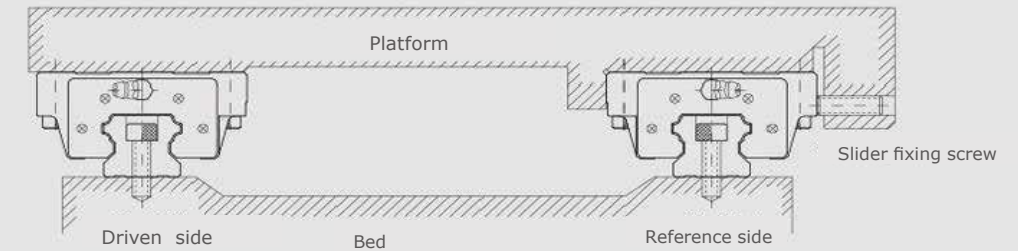
1.10.1 Reference Rail and Driven Rail



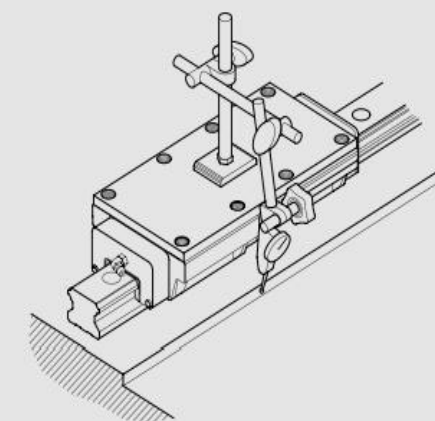
Use a Straight Edge

Position a straight edge between the two rails then confirm parallelism with a dial gauge. Using the straight edge as a reference to confirm subsidiary rail straightness from one end to the other, tightening the mounting bolts in sequence as you go.

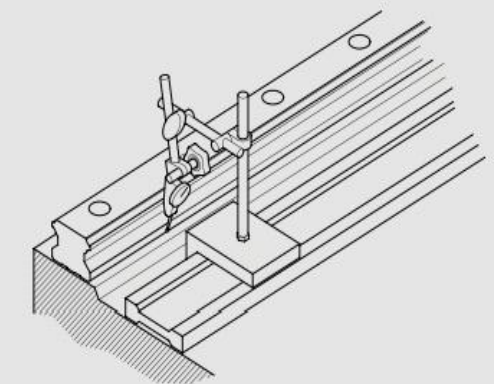
1.10.4 Installation of the Linear Guide without a Reference Section for the Master Linear Guide Rail.



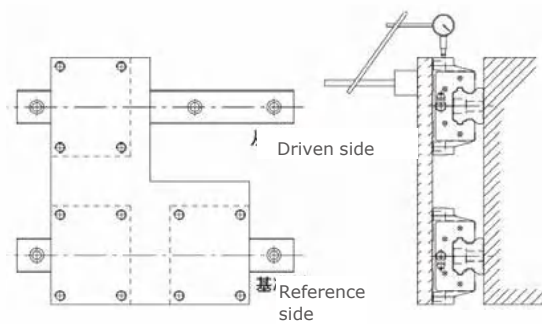
Installation of the reference side guide



○ Mounting the Master Linear Guide Rail Use a Temporary Reference Surface from end to end to acquire Linear Guide rail straightness. For this method, however, two Linear Guide blocks must be fastened together, positioned on the top of each other while attached to a measurement plate

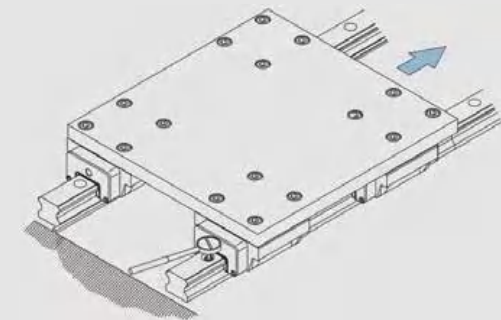


○ Use a Straight Edge After temporarily tightening the mounting bolts, use a dial gauge to check the straightness of the Linear Guide-rail-side reference surface from end to end, fully tightening the mounting bolts in sequence as you go. To mount the subsidiary Linear Guide rail, follow the procedures specified in the second paragraph on the previous page.



○ Move the Table

Fasten two Linear Guide blocks on the master side to the table (or a temporary measurement table). Temporarily fasten the subsidiary Linear Guide rail and block to the base and table. From the dial-gauge stand, with a dial gauge contact the subsidiary rail Linear Guide block side, move the table from the rail end and check the parallelism between the block and the subsidiary Linear Guide rail, fastening the bolts on sequences as you go.



○ Compare to the Master Linear Guide Rail

Make sure the master Linear Guide rail is properly installed. Temporarily fasten the subsidiary Linear Guide rail in place. Place a table on the Linear Guide blocks mounted on the master rail and on the temporarily fastened subsidiary

2.1 SG Series Heavy-Duty Ball Linear Guide

The SG series linear guides an effort to meet customer's requirement, DTX-LG offers several different types of guides. Except for DTX-LG develop SG series with self lubrication system which is designed for environment with high contamination and series for small machines and semiconductor industry.

2.1.1 SG Series Linear Guide Features

(1) Self-aligning capability

The combination of DF (45°-45°) fom the are groove allows for selfaligning capability during installation, Through the elastic deformation of steel balls and the transfer of contact points, even ifthe installation surface is slightly misaligned, it can be absorbed by the internal linear guide slider, producing a self-aligning effect and achieving high-accuracy, stable, and smooth movement.

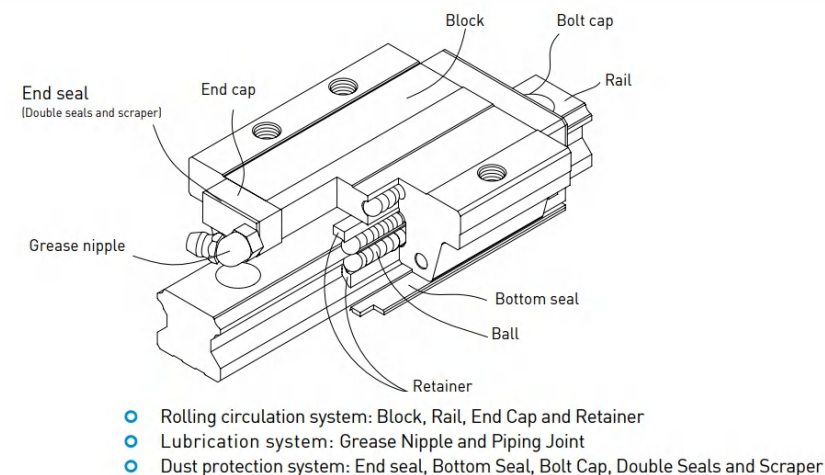
(2) Interchangeability

Due to strict control of manufacturing accuracy, the dimensions of linear guides can be maintained at a certain level, and the sliders are designed with retainers to prevent the steel balls from falling out, Therefore, some series have interchangeable accuracy, and customers can order guides or sliders as needed, or store guides and sliders separately to reduce storage space.

(3) High rigidity in all directions

The use of four rows of circular are grooves, combined with four rows of steel balls with a contact angle of 45 degrees, allows the steel balls to achieve an ideal two-point contact structure that can withstand loads from above, below, and both sides. Ifnecessary, preload can be applied to increase rigidity.

2.1.2 The Structure of SG Series



2.1.3 The Characteristics of SG Series

Smooth Movement- DTX circulation system of Linear Guide block is designed to perform smooth movement.
High Stability/Rigidity- Linear Guide block is designed under DTX's exclusive patent that can increases depth of material to improvethe strength capacity, prevent deflection and provide high rigidity.
DTX the exclusive contact point design promotes high rigidity, Moreover, self-aligning balance

SG Series

Heavy-Duty Ball Linear Guide

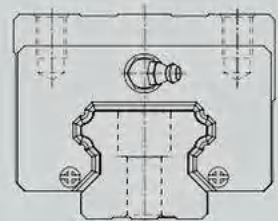
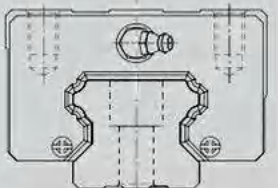
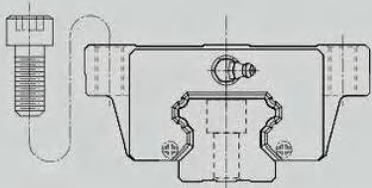
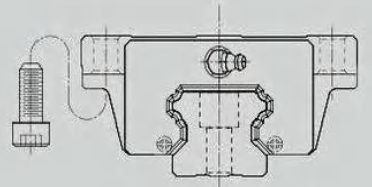
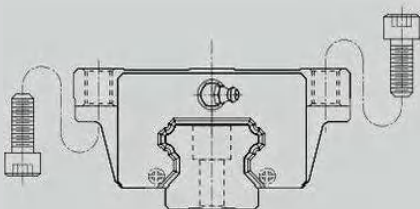
2.1.4 SG Series Model

(1) Slider types

DTX-LG offers two types of linear guides: flange type and square type. The square type linear guides are divided into H-type and L-type.

The L-type is a low-assembly linear guide based on the H-type, with the same assembly height as the flange type linear guide.

Table 2.1.1 Slider types

Type	Spec	Shape	Height (mm)	Rail length (mm)	Equipment
Square type	SGH-CA SGH-HA		28 ↓ 90	100 ↓ 4000	- Machining centers - Machine tools - Accuracy machining equipment - Heavy-duty cutting machines - Marble cutting machines - Grinding machines - Injection molding machines - Presses - Automation equipment - Transportation equipment - Measuring instruments
	SGL-CA SGL-HA		24 ↓ 70	100 ↓ 4000	
Flange type	SGW-CA SGW-HA		24 ↓ 90	100 ↓ 4000	
	SGW-CB SGW-HB		24 ↓ 90	100 ↓ 4000	
	SGW-CC SGW-HC		24 ↓ 90	100 ↓ 4000	

Mounting Type of Linear Rail

SG Series Heavy-Duty Linear Guide

Table 2-1-5 Assembly accuracy table

Unit: mm

Model	SG - 45, 55				
Accuracy grade	Normal grade [C]	High grade [H]	Accuracy grade [P]	Supper grade [SP]	Ultra grade [UP]
Dimensional tolerance of height H	±0.1	±0.05	$\begin{matrix} 0 \\ -0.05 \end{matrix}$	$\begin{matrix} 0 \\ -0.03 \end{matrix}$	$\begin{matrix} 0 \\ -0.02 \end{matrix}$
Dimensional tolerance of width N	±0.1	±0.05	$\begin{matrix} 0 \\ -0.05 \end{matrix}$	$\begin{matrix} 0 \\ -0.03 \end{matrix}$	$\begin{matrix} 0 \\ -0.02 \end{matrix}$
Variation of height H	0.03	0.015	0.007	0.005	0.003
Variation of height H	0.03	0.02	0.01	0.007	0.005
Running parallelism of block surface C to surface A	Traveling parallelism (see Table 2.1.11)				
Running parallelism of block surface D to surface B	Traveling parallelism (see Table 2.1.11)				

Table 2-1-6 Assembly accuracy table

Unit: mm

Model	SG - 65				
Accuracy grade	Normal grade [C]	High grade [H]	Accuracy grade [P]	Supper grade [SP]	Ultra grade [UP]
Dimensional tolerance of height H	±0.1	±0.07	$\begin{matrix} 0 \\ -0.07 \end{matrix}$	$\begin{matrix} 0 \\ -0.05 \end{matrix}$	$\begin{matrix} 0 \\ -0.03 \end{matrix}$
Dimensional tolerance of width N	±0.1	±0.07	$\begin{matrix} 0 \\ -0.07 \end{matrix}$	$\begin{matrix} 0 \\ -0.05 \end{matrix}$	$\begin{matrix} 0 \\ -0.03 \end{matrix}$
Variation of height H	0.03	0.02	0.01	0.007	0.005
Variation of height H	0.03	0.025	0.015		

SG Series

Heavy-Duty Ball Linear Guide

2-1-6 Preload

(1) Determining the Magnitude of a Preload

Using larger rolling elements helps strengthen the entire rigidity of the block while there exists clearance within ball circulation. Increasing preload would decrease the vibration and reduce the corrosion caused by running back and forth. However, it would also add the workload within those rolling elements. The greater the preload is, the greater the inner workload is. Therefore, choosing preload has to consider the effect carefully between vibration and preload.

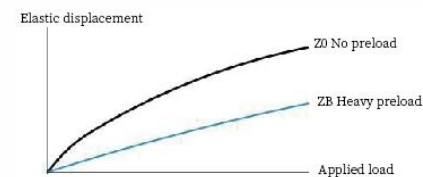
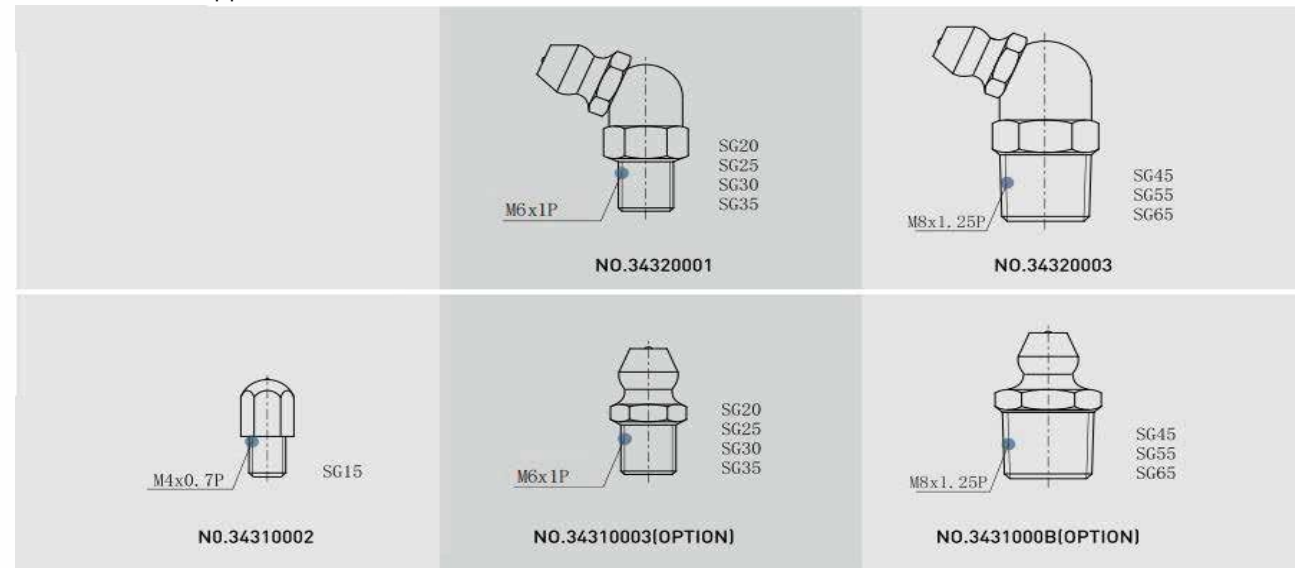


Table 2-1-12 Preload grades

Preload grade	Mark	Preload	Conditions of use	Scope of application
No preload	Z0	0-0.02C	Fixed load direction and low impact, low accuracy requirements	Conveyor systems, automatic packaging machines, automated industrial machinery, XY axes for general industrial machinery, welding machines, fusing machines, tool exchange devices
Medium preload	ZA	0.05-0.07C	Light load and high accuracy requirements	Z-axis of general industrial machinery, electrical discharge machines, NC lathes, precision XY platforms, measuring instruments, machining centers, vertical machining centers, industrial robots, automatic coating machines, and various high-speed material supply devices.
Heavy preload	ZB	0.10-0.12C	Rigidity requirements, vibration and impact in the operating environment	Machining centers, grinding machines, NC lathes, vertical or horizontal milling machines, Z-axis of machine tools, heavy cutting machines
Grade	Interchangeable linear guide (single part)		Non-interchangeable linear guide (combined parts)	
Preload grade	Z0, ZA		Z0, ZA, ZB	

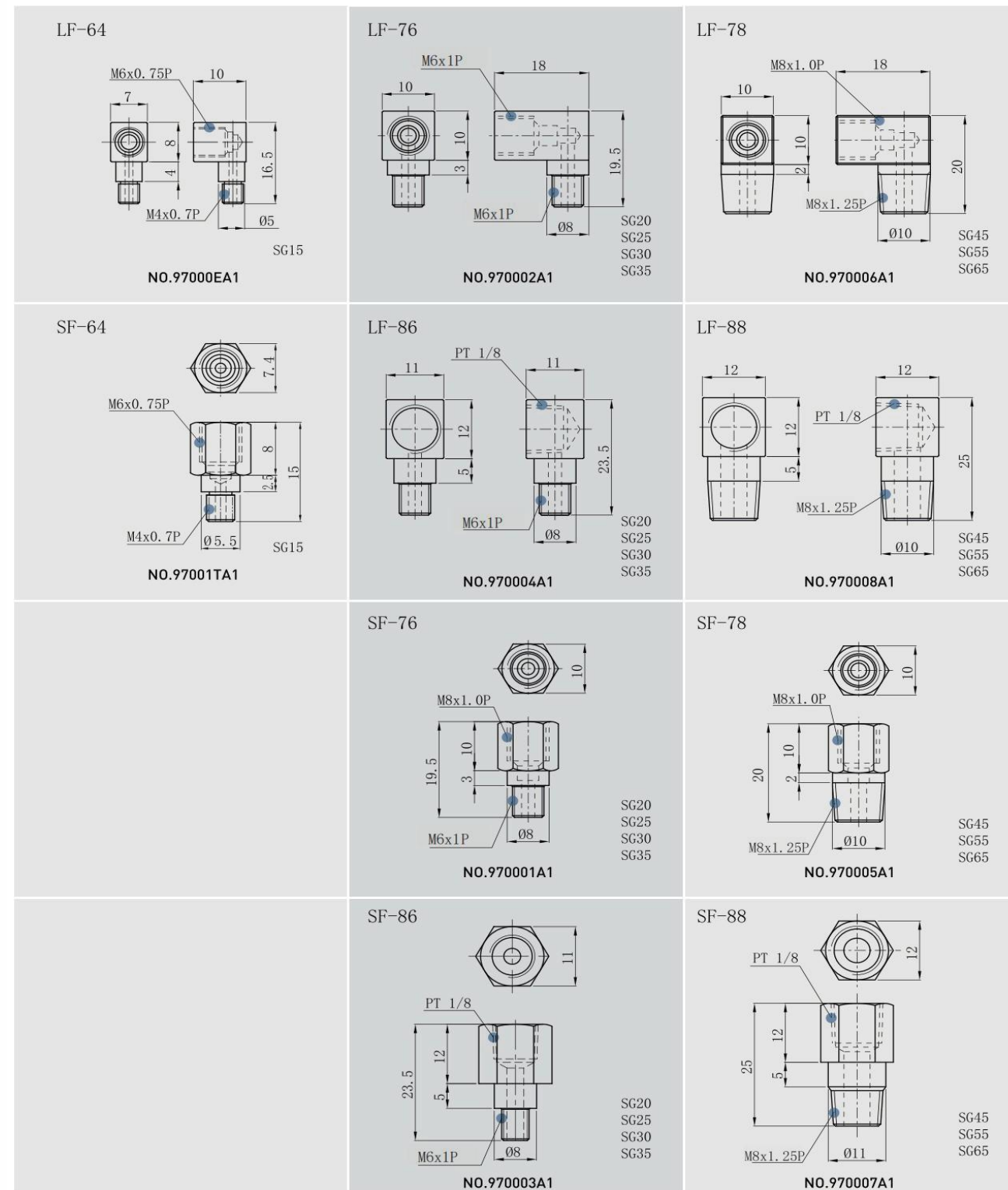
Note: Preload C is the dynamic rated load.

2-1-7 Grease Nipples



SG Series Heavy-Duty Ball Linear Guide

Types of Lubrication Coupler



Oil supply rate

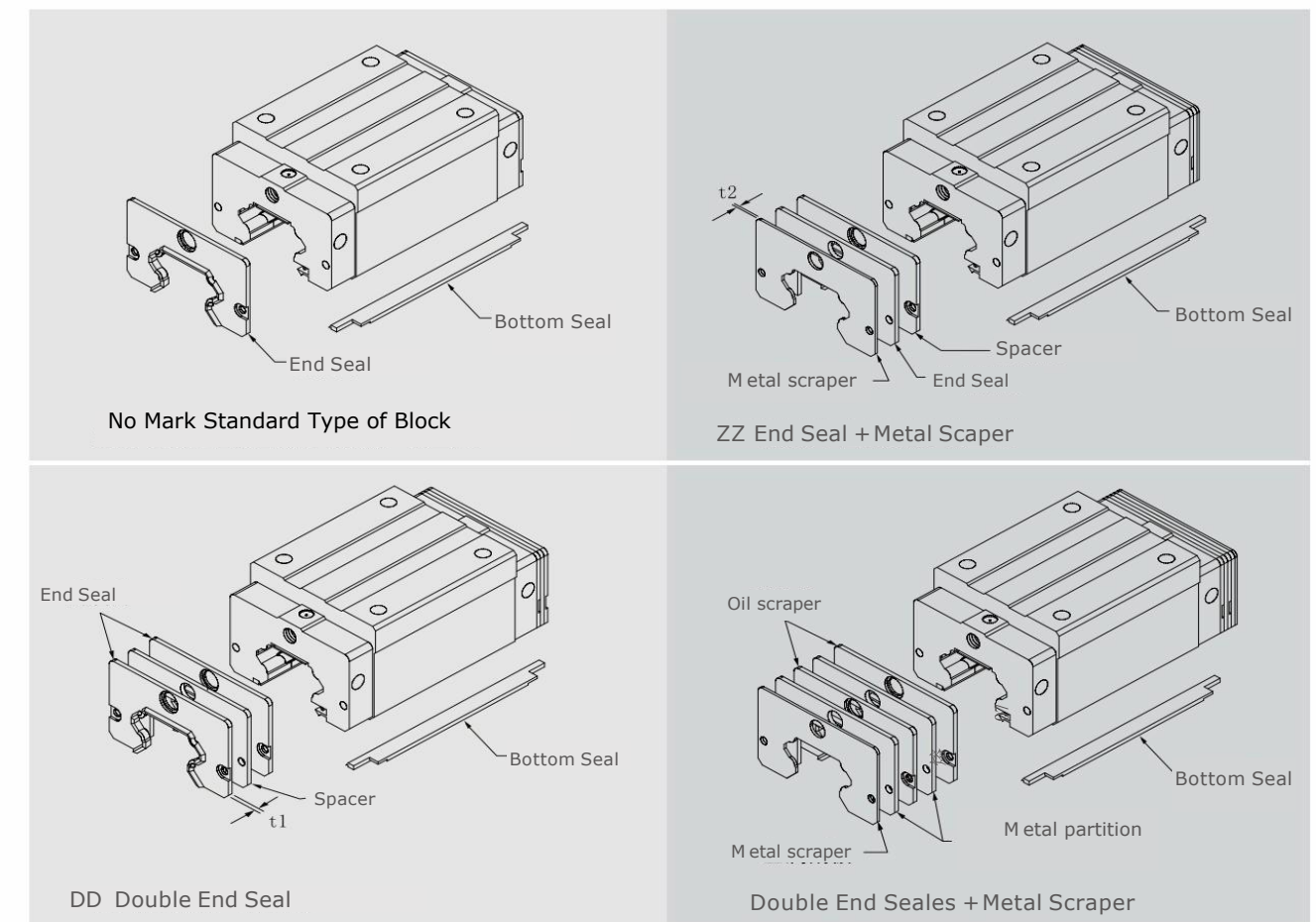
Table 2-1-15 Oil supply rate

Specification	Oil supply rate	Specification	Oil supply rate
SG 15	0.2	SG 35	0.3
SG 20	0.2	SG 45	0.4
SG 25	0.3	SG 55	0.5
SG 30	0.3	SG 65	0.6

2.1.8 Dust protection

(1) Standard dust protection equipment code

For use in general operating environments without special requirements. If you require the following dust-proof accessories, please add the code after the product model number.



SG Series HHeavy-Duty Ball Linear Guide

● Rail Cap

Once the Linear Guide is operating in a cutting machine, dust and foreign matter that enter the Linear Guide may cause abnormal wear and shorten the service life.

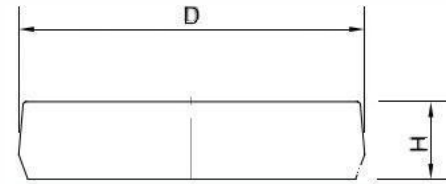


Table 2.1-18

Guide specification	Installation screw	Diameter (D) [mm]	Thickness (H) [mm]	Guide specification	Installation screw	Diameter [D] [mm]	Thickness (H) [mm]
SGR15	M4	7.65	1.1	SGR35	M8	14.20	3.5
SGR20	M5	9.65	2.5	SGR45	M12	20.25	4.5
SGR25	M6	11.15	2.5	SGR55	M14	23.25	5.0
SGR30	M8	14.20	3.5	SGR65	M16	26.35	5.0

(5)Dust Proof Accessories of Total slider length

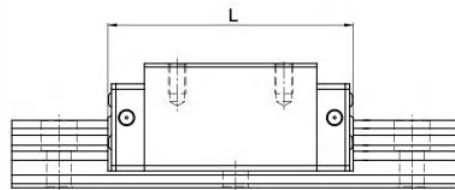


Table 2-1-19 Dust protection code for total slider length

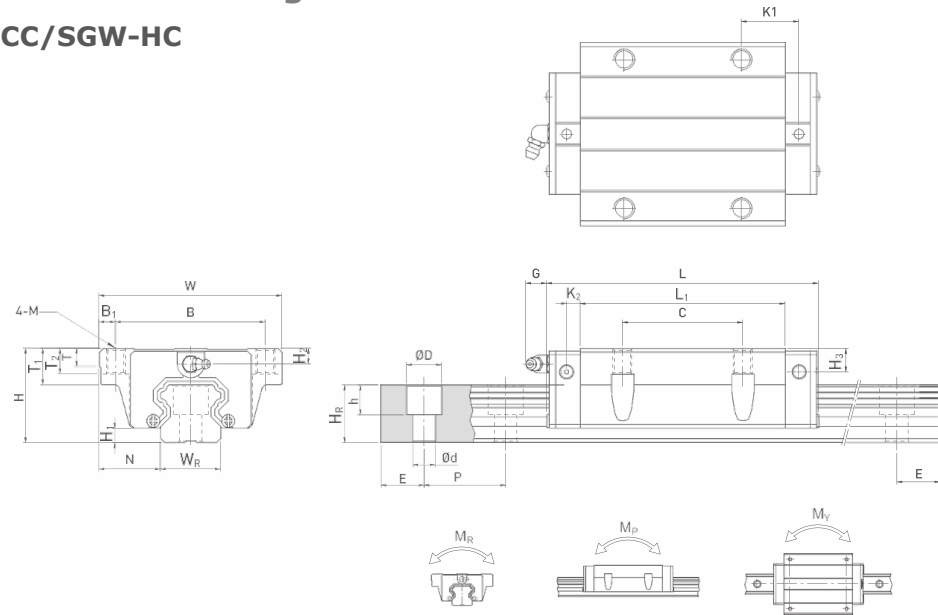
Unit: mm

Specification	Dust protection code for total slider length (L)					
	SS/SH	ZZ/ZH	DD/DH	KK/KH	SW	ZW
SG15C	61.4(618)	69.0 (69.4)	68.0 (68.4)	75.6 (76.0)	63.2 (63.2)	71.0(71.4)
*SG20C	77.5 (79.3)	82.5 (84.5)	82.5 (84.3)	87.5 (89.5)	78.5 (79.3)	86.3 (88.3)

SG Series

Heavy load ball linear guide

(5)SGW-CC/SGW-HC

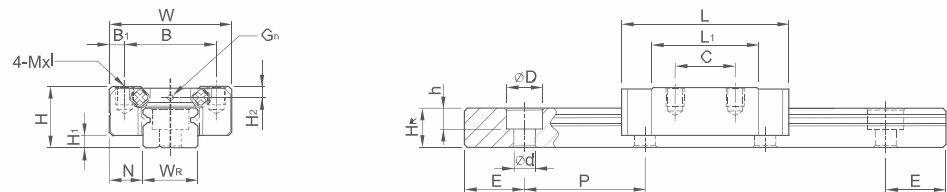


Model	Component size (mm)			Slider size (mm)															Guide size (mm)					Guide fixing bolt size	Basic dynamic rated load C(kN)	Basic static rated load C ₀ (kN)	Permissible static moment			Weight	
	H	H ₁	N	W	B	B ₁	C	L _t	L	K ₁	K ₂	G	M	T	T ₁	H ₂	H ₃	W _R	H _R	D	h	d	P	E	(mm)	C(kN)					

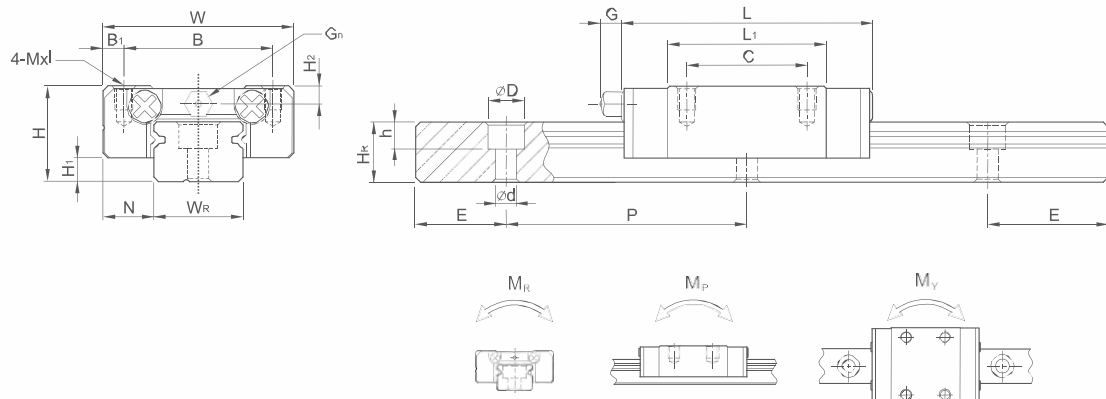
SMN/SMW Series linearguide size table

(1) SMN-C/SMN-H

SMN7,SMN9,SMN12



SMN15



Model	Component size (mm)		Slider size (mm)										Guide size (mm)										Permissible static moment			Weight	
	H	H ₁	N	W	B	B ₁	C	L ₁	L	G	G _n	M×I	H ₂	W _R	H _R	D	h	d	P	E	(mm)	C(kN)	C ₀ (kN)	MR N-m	MP N-m	MY N-m	Slider kg
SMN7C	8	1.5	5	1	7	2	5																				

SR Series Roller Linear Guide

Dust proof Description

End Seal and Bottom Seal

To prevent life reduction caused by iron chips or dust entering the block.

Double Seal

Enhances the wiping effect, foreign matter can be completely wiped off.

Table 2.4.18 Dimension of Seal

Size	Thickness(t1) (mm)	Size	Thickness(t1) (mm)
SR15ES	2.2	SR35ES	2.5
SR20ES	2.2	SR45ES	3.6
SR25ES	2.2	SR55ES	3.6
SR30ES	2.4	SR65ES	4.4

(4) Scraper

The scraper removes high-temperature iron chips and larger foreign objects.

Table 2.4.19 Dimensions of Scraper

Size	Thickness(t2) (mm)	Size	Thickness(t2) (mm)
SR15SC	1.0	SR35SC	1.5
SR20 SC	1.0	SR45SC	1.5
SR 25 SC	1.0	SR55SC	1.5
SR30 SC	1.5	SR65SC	1.5

(5) Bolt caps for rail mounting holes

Caps are used to cover the mounting holes to prevent chips or other foreign objects from collecting in the holes. The caps will be enclosed in each rail package

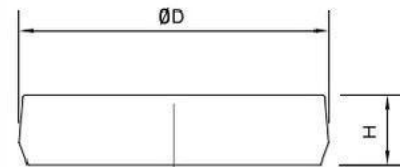
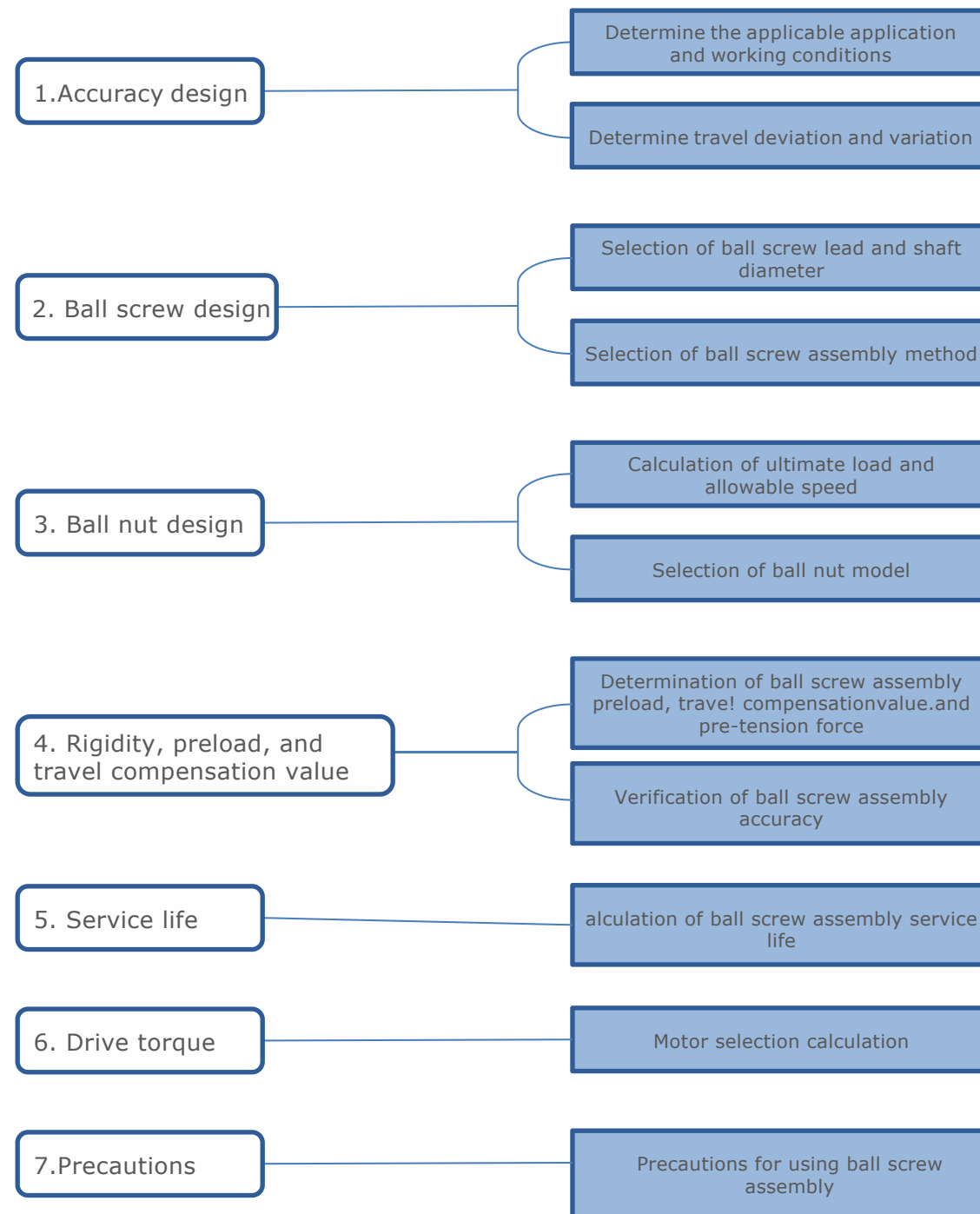


Table 2.4.20 Dimensions of bolt caps for rail mounting holes

Rail size	Bolt Size	Diameter(D) (mm)	
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Ball screw assembly selection flowchart



Accuracy Selection

Among the factors that cause feed accuracy errors, lead accuracy and feed system rigidity are the key points for review, while other factors such as heat deformation due to temperature rise as well as assembly accuracy for the guiding surface, etc. should also be considered.

Application		Shaft	Accuracy grade					
			◇	◇	◇	◇	◇	◇
CNC machines	Lathe	X	○	○	○	○	○	○
		Z			○	○	○	○
	Milling machine Boring machine	x、Y						

